

Student Name:

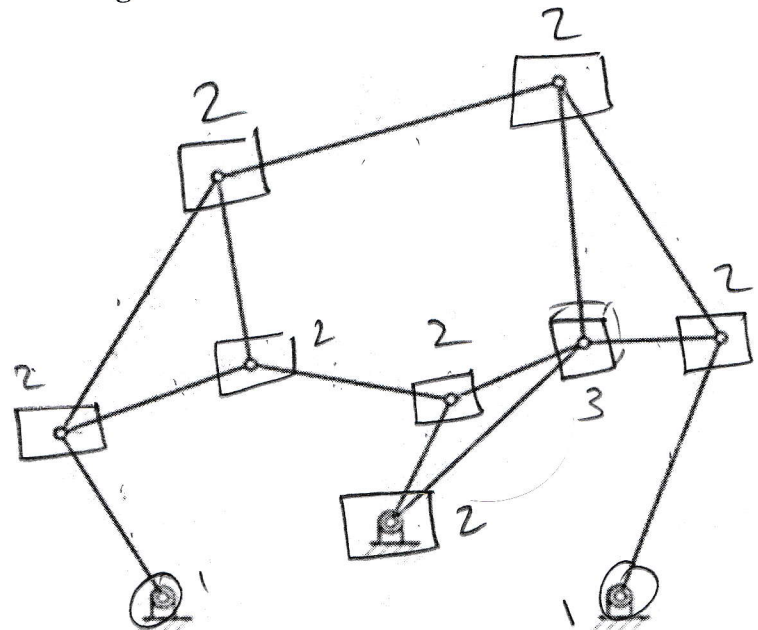
Student ID number:

Problem: Determine the mobility for the following mechanism

$$n = 14$$

$$P_1 = 14$$

$$P_2 = 0$$



$$M = 3(n-1) - 2P_1 - P_2 = 3(14-1) - (2)(14) - 0$$

$$\Rightarrow M = 1$$

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Problem: Determine the mobility for the following mechanism

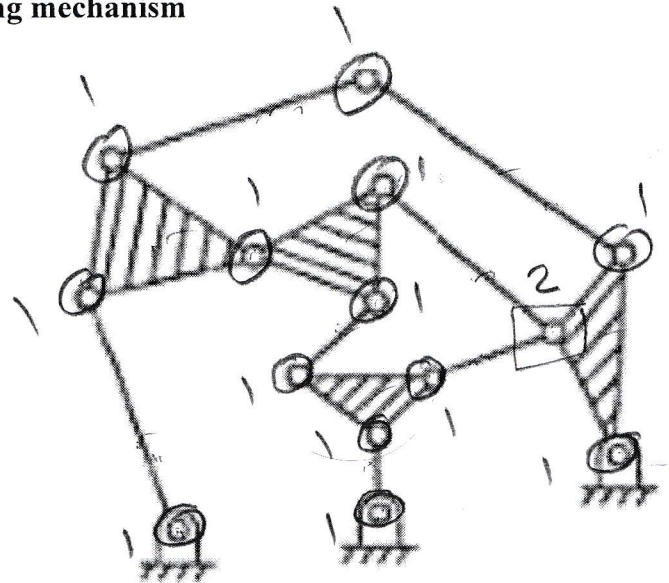
$$n = 12$$

$$P_1 = 15$$

$$P_2 = 0$$

$$M = 3(n-1) - 2P_1 - P_2$$

$$\Rightarrow M = 3(12-1) - 2(15) - 0 = \boxed{3}$$



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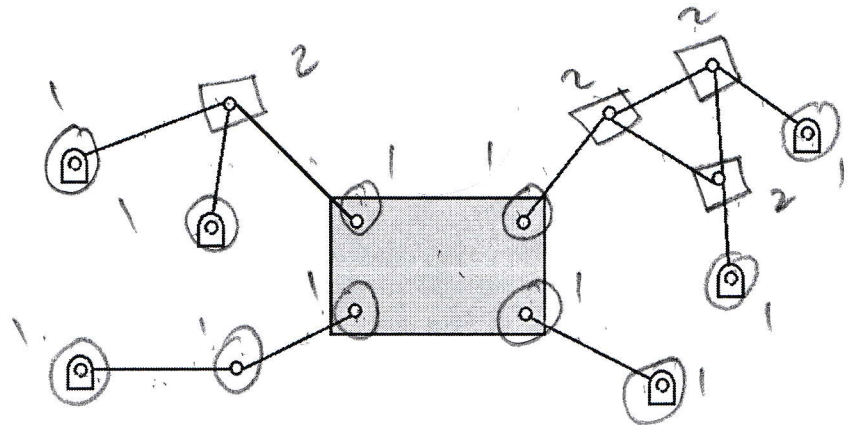
Student ID number:

Problem: Determine the mobility for the following mechanism

$$n = 14$$

$$P_1 = 19$$

$$P_2 = 0$$



$$M = 3(n-1) - 2P_1 - P_2 = 3(14-1) - (2)(19) - 0 = 1$$

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Consider the following loop closer equation for a four bar mechanism:

$$d_2 U_{\theta_2} + d_3 U_{\theta_3} - d_1 U_{\theta_1} = d_4 U_{\theta_4}$$

Assume link (1) $d_1 U_{\theta_1}$ is ground. Find ω_3 and ω_4 if

$\theta_1 = 0$	$\theta_2 = 60$ degree	$\theta_3 = 30$ degree	$\theta_4 = 90$ degree	$\omega_2 = 600$ RPM
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$$d_1 = 1.2 \text{ m} \quad d_2 = 0.2 \text{ m} \quad d_3 = 0.8 \text{ m} \quad d_4 = 0.4 \text{ m}$$

Sol:-

$$\frac{d}{dt} [d_2 U_{\theta_2} + d_3 U_{\theta_3} - d_1 U_{\theta_1} = d_4 U_{\theta_4}]$$

$$d_2 \omega_2 U_{\theta_2} + d_3 \omega_3 U_{\theta_3} - 0 = d_4 \omega_4 U_{\theta_4} \quad \text{--- (1)}$$

To eliminate ω_4 , dot Eq (1) with U_{θ_4} :

$$d_2 \omega_2 \sin(\theta_4 - \theta_2) + d_3 \omega_3 \sin(\theta_4 - \theta_3) = 0$$

$$\Rightarrow \omega_3 = - \frac{d_2 \omega_2 \sin(\theta_4 - \theta_2)}{d_3 \sin(\theta_4 - \theta_3)} = - \frac{(0.2)(600) \sin(90 - 60)}{0.8 \sin(90 - 30)}$$

$$\Rightarrow \omega_3 = - 86.6 \text{ RPM}$$

to find ω_4 dot Eq 1 by U_{θ_3}

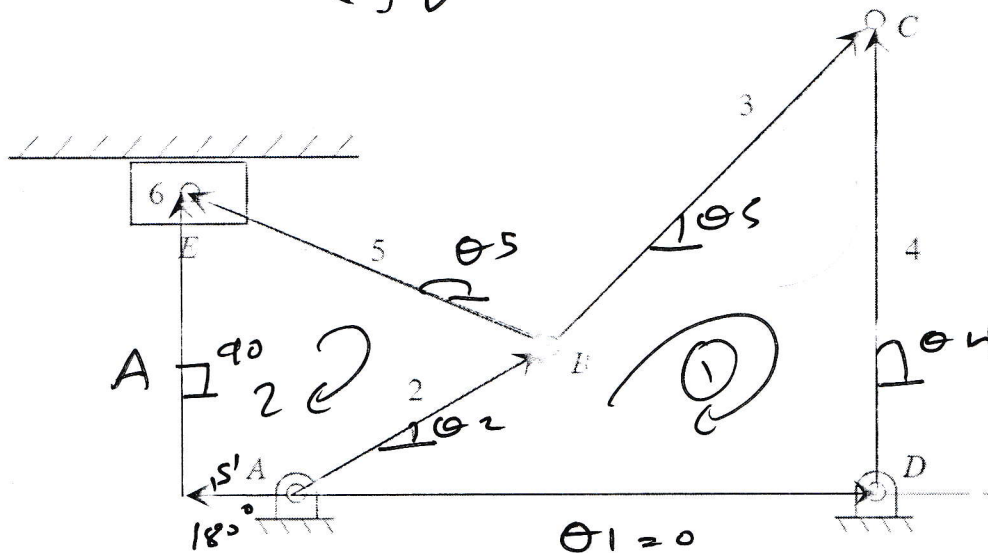
$$d_2 \omega_2 \sin(\theta_3 - \theta_2) + 0 = d_4 \omega_4 \sin(\theta_3 - \theta_4)$$

$$\omega_4 = \frac{d_2 \omega_2 \sin(\theta_3 - \theta_2)}{d_4 \sin(\theta_3 - \theta_4)} = \frac{0.2 (600) \sin(30 - 60)}{0.4 \sin(30 - 90)} = 173.2 \text{ RPM}$$

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Problem: derive the acceleration loop equation for the following mechanism



Loop 1:-

$$d_1 u_{\theta_1} + d_4 u_{\theta_4} = d_2 u_{\theta_2} + d_3 u_{\theta_3} \quad \text{--- (1)}$$

$$d_2 u_{\theta_2} + d_5 u_{\theta_5} = \delta u_{180} + A u_{90} \quad \text{--- (2)}$$

$$\frac{d}{dt} \text{ (2) } \& \text{ (Eq 2):}$$

$$0 + d_4 [\dot{\theta}_4 u_{\theta_4} - \omega_4^2 u_{\theta_4}] = d_2 [\dot{\theta}_2 u_{\theta_2} - \omega_2^2 u_{\theta_2}] + d_5 [\dot{\theta}_5 u_{\theta_5} - \omega_5^2 u_{\theta_5}] \quad \text{--- (3)}$$

$$d_2 [\dot{\theta}_2 u_{\theta_2} - \omega_2^2 u_{\theta_2}] + d_5 [\dot{\theta}_5 u_{\theta_5} - \omega_5^2 u_{\theta_5}] = \ddot{\delta} u_{180} \quad \text{--- (4)}$$

Eqs (3) & (4) are the loop closure Eqs for acc.

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Problem: a cam segment has the following specifications

- Rise from 0 to 3 cm for a period from 0 to 120° degree
- The rise is a parabolic

typical

Find the follower travel at $\theta = 80^\circ$

Because $\beta = 120^\circ$ & $\theta = 80^\circ$

$$S = H \left[1 - 2 \left(1 - \frac{\theta}{\beta} \right)^2 \right]$$

$$S = 3 \left[1 - 2 \left(1 - \frac{80}{120} \right)^2 \right]$$

$$= \underline{2.33 \text{ cm}}$$

$$S = 2H \left(\frac{\theta}{\beta} \right)^2; S = H \left(1 - 2 \left(1 - \frac{\theta}{\beta} \right)^2 \right)$$

$$s(\theta) = \frac{H}{2} \left(1 - \cos \left(\frac{\pi\theta}{\beta} \right) \right)$$

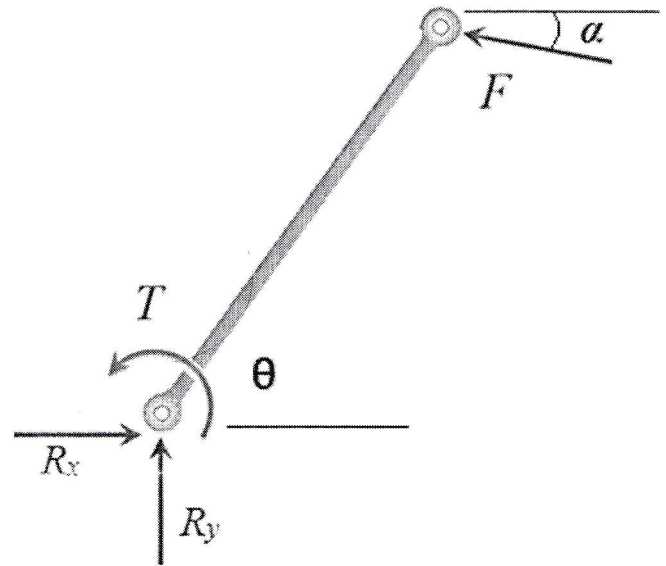
$$s(\theta) = \frac{h}{\pi} \left(\frac{\pi\theta}{\beta} - \frac{1}{2} \sin \left(\frac{2\pi\theta}{\beta} \right) \right)$$

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Problem: consider the following bar link. If its length is 40cm, $\theta = 60$ degree, $\alpha = 30$ degree and $T = 50$ N.M.

Find the reactions R_x , R_y and F .



$$\textcircled{1} \rightarrow \sum F_x = 0$$

$$\Rightarrow R_x - F \cos \alpha = 0$$

$$\Rightarrow R_x - F \cos 30 = 0 \quad \text{--- (1)}$$

$$\textcircled{2} \uparrow \sum F_y = 0$$

$$\Rightarrow R_y + F \sin \alpha = 0$$

$$\Rightarrow R_y + F \sin 30 = 0$$

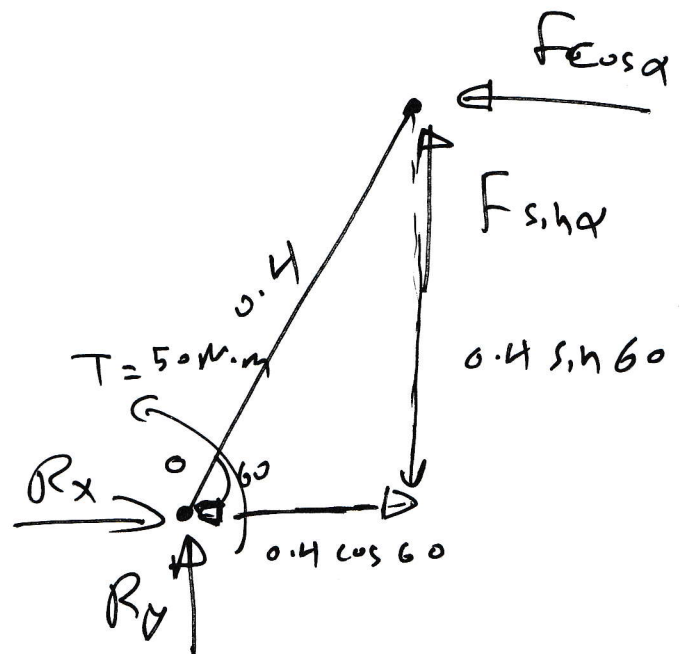
$$\textcircled{3} \uparrow \sum M_o = 0$$

$$T + F \cos \alpha (0.4 \sin 60) + F \sin \alpha (0.4 \cos 60) = 0$$

$$\Rightarrow 50 + F [0.3 + 0.1] = 0 \Rightarrow F = -125 \text{ N}$$

$$R_x = -108.3 \text{ N}$$

$$R_y = 62.5 \text{ N}$$



Faculty of Engineering	Philadelphia University	Mechanical Eng. Dep.
Course name: Theory of machines	4 th and 5 th Quiz	Course number: 620333 class(1)
Instructor: Eng. Laith Batarseh	Thursday 9/1/2018	Allowed time: 20 minutes

Student Name:

Student ID number:

Problem 1: a cam segment has the following specifications

- Rise from 0 to 5cm for a period from 0 to 90° degree
- The rise is a parabolic

Find the follower travel at $\theta = 40^\circ$

$$\beta = 90 \quad \beta/2 = 45$$

So θ lies in the period

$$0 \leq \theta \leq \frac{\beta}{2}$$

$$S = 2H \left(\frac{\theta}{\beta} \right)^2$$

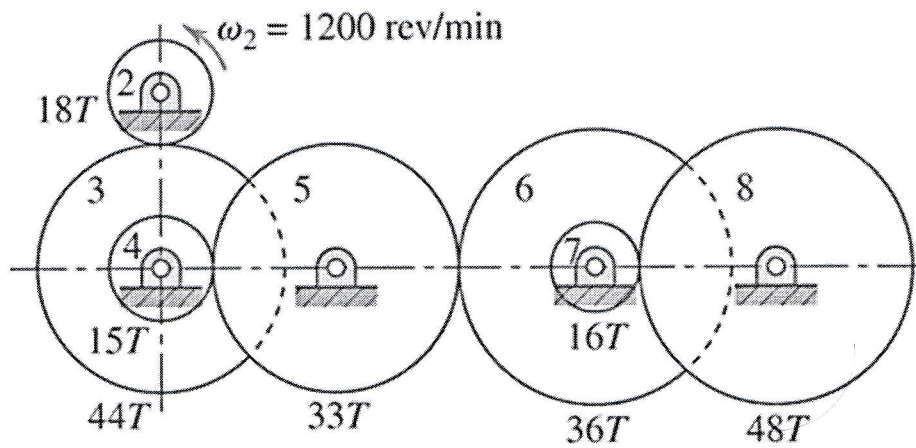
$$= 2(5) \left(\frac{40}{90} \right)^2 = 1.97 \text{ cm}$$

$$S = 2H \left(\frac{\theta}{\beta} \right)^2; S = H \left(1 - 2 \left(1 - \frac{\theta}{\beta} \right)^2 \right)$$

$$s(\theta) = \frac{H}{2} \left(1 - \cos \left(\frac{\pi\theta}{\beta} \right) \right)$$

$$s(\theta) = \frac{h}{\pi} \left(\frac{\pi\theta}{\beta} - \frac{1}{2} \sin \left(\frac{2\pi\theta}{\beta} \right) \right)$$

Problem 2: For the following compound gear train, find ω_8 . T stand for teeth



$$\frac{\omega_8}{\omega_2} = \left[-\frac{\omega_3}{\omega_2} \right] \left[\frac{\omega_4}{\omega_3} \right] \left[-\frac{\omega_5}{\omega_4} \right] \left[-\frac{\omega_6}{\omega_5} \right] \left[\frac{\omega_7}{\omega_6} \right] \left[-\frac{\omega_8}{\omega_7} \right]$$

$$= + \frac{\omega_3 \omega_5 \omega_6 \omega_8}{\omega_2 \omega_4 \omega_5 \omega_7}$$

$$= \frac{N_2 N_4 N_7}{N_3 N_6 N_8} = \frac{(18)(15)(16)}{(44)(36)(48)}$$

$$\frac{\omega_8}{\omega_2} = \frac{5}{88} \rightarrow \omega_8 = 1200 \left(\frac{5}{88} \right) = 68.2 \text{ RPM}$$

Faculty of Engineering	Philadelphia University	Mechanical Eng. Dep.
Course name: Theory of machines	4 th and 5 th Quiz	Course number: 620333 class(2)
Instructor: Eng. Laith Batarseh	Monday 15/1/2018	Allowed time: 20 minutes

Student Name:

Student ID number:

Problem 1: if $\omega_2 = 400$ RPM and $\omega_{arm} = -100$ for the gear train. Find the speed of gear 6.

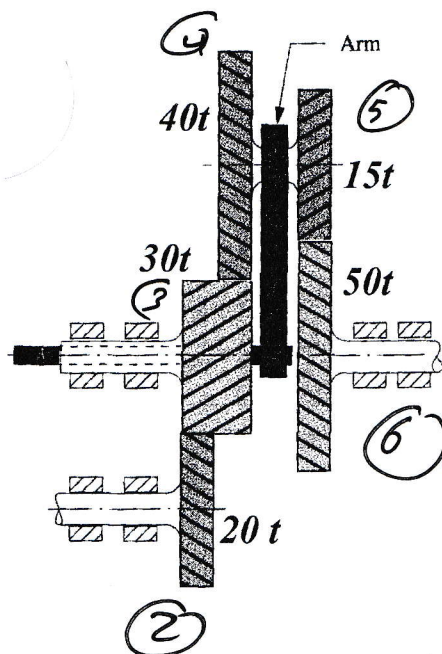
$$\frac{\omega_6 - \omega_a}{\omega_2 - \omega_a} = \left[-\frac{\omega_3}{\omega_2} \right] \left[-\frac{\omega_4}{\omega_3} \right] \left[\frac{\omega_5}{\omega_4} \right] \left[-\frac{\omega_6}{\omega_5} \right]$$

$$\Rightarrow \frac{\omega_6 + 100}{400 + 100} = - \frac{\omega_4 \omega_6}{\omega_2 \omega_5}$$

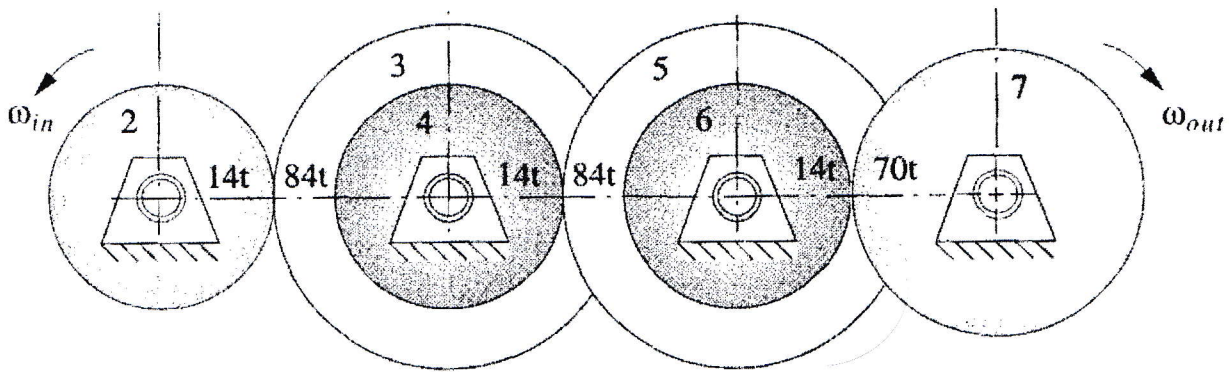
$$\Rightarrow \frac{\omega_6 + 100}{500} = - \frac{N_2 N_5}{N_4 N_6}$$

$$\omega_6 = -500 \left[\frac{(20)(15)}{(40)(50)} \right] - 100$$

$$\omega_6 = -175 \text{ RPM} = 175 \text{ RPM} \curvearrowright$$



Problem 2: consider the following **compound** gear train, if $\omega_2 = 600$ RPM, find ω_7 . t stand for teeth



$$\frac{\omega_7}{\omega_2} = \left[-\frac{\omega_3}{\omega_2} \right] \left[\frac{\omega_4}{\omega_3} \right] \left[-\frac{\omega_5}{\omega_4} \right] \left[\frac{\omega_6}{\omega_5} \right] \left[-\frac{\omega_7}{\omega_6} \right]$$

$$\frac{\omega_7}{\omega_2} = - \frac{\omega_3 \omega_5 \omega_7}{\omega_2 \omega_4 \omega_6} = - \frac{N_2 N_4 N_6}{N_3 N_5 N_7}$$

$$\omega_7 = \omega_2 \frac{N_2 N_4 N_6}{N_3 N_5 N_7} = 600 \left[\frac{(14)(14)(14)}{(84)(84)(70)} \right]$$

$$\omega_7 = - 3.33 \text{ RPM.}$$