



Student Name:  
Student Number:  
Serial Number:

Second exam, second semester: 2016/2017  
Mechanical Engineering Department

Course Title: theory of machines  
Course No: 620333  
Lecturer: Eng. Laith Batarseh

Date: 2 /1/2017  
Time: 50min  
No. of pages: 4

Sun, Thu and Tue 12:10 – 1:00

Mon and Wed 12:45 – 2:15

Instructions:

- **ALLOWED:** Non-programmable calculator, pens and drawing tools (**no red color**).
- **NOT ALLOWED:** Papers, literatures and any handouts. Otherwise, it will lead to the non-approval of your examination.
- **Shut down** Telephones, and other communication devices.

Please note:

- Write your name and your matriculation number on every page of the solution sheets.
- All solutions together with solution methods (explanatory statement) must be inserted in the labeled position on the solution sheets.
- Support your answer with diagrams, equations and examples when possible
- You can submit your exam after the first ½ hour.
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**Question 1:**

**(8marks)**

*General concepts*

Chose the correct answer for the short questions in page 2 and fill the table given in page 1 of 4 with your answers. Use the symbol (X).

Ans/Q	1	2	3	4	5	6	7	8
<i>a</i>		X						
<i>b</i>	X				X		X	X
<i>c</i>				X		X		
<i>d</i>			X					
<i>e</i>								



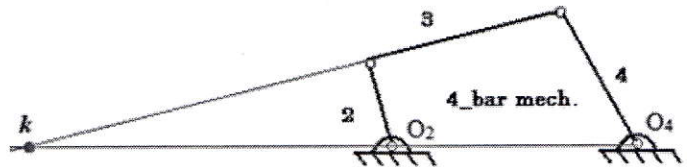
**Question 1:** *continu*

**(8marks)**

1. A mechanism has 4 links will has a number of instantaneous centers equal

- a. 15      **b. 6**      c. 21      d. 10      e. none of the previous

2. For the four bar mechanism shown in the fig. if  $\omega_2 = 1200$  RPM,  $O_2k = 40$ cm and  $O_4k = 80$ cm. find the angular speed of link 4 ( $\omega_4$ ) in RPM



- a. 600**      b. 150      c. 300      d. 1200      e. none of the previous

3. A rigid bar link has length equal 50cm and rotates about one of its joints by angular speed equal 50 rad/s. its tangential velocity in m/s will equal

- a. 100      b. 50      c. 200      **d. 25**      e. none of the previous

4. a 1.5m bar link rotate about one of its joints at a constant angular speed equal 10 rad/s. then the magnitude of its acceleration equal in  $m/s^2$ :

- a. 3750      b. 1350      **c. 150**      d. 600      e. none of the previous

5. a bar link is analyzed statically. If the force at one of its joints is  $-5i + 7j$ , the Cartesian vector for the force in the other joint will be

- a.  $-5i + 7j$       **b.  $5i - 7j$**       c.  $-7i + 5j$       d.  $7i - 5j$       e. none of the previous

The flowing data are for questions 6 and 7:

When a cam rotates with speed equal 600 RPM from  $0^\circ$  to  $90^\circ$  it makes the follower to raise in SHM from 0cm to 4cm.

6. Find the follower displacement at cam angle equal  $45^\circ$ .

- a. 3.72 cm      b. 3.00 cm      **c. 2.00 cm**      d. 1.00 cm      e. none of the previous

7. Find the follower velocity at cam angle equal  $45^\circ$ . round the solution to two decimal digits.

- a. 3.15 m/s      **b. 2.51 m/s**      c. 2.17 m/s      d. 1.26 m/s      e. none of the previous

8. An eccentric cam has eccentricity ( $e$ ) = 10 cm, find the displacement at angle =  $180^\circ$ .

- a. 15.0 cm      **b. 20.0 cm**      c. 13.4 cm      d. 5.0 cm      e. none of the previous

**This space can be used for the calculations in problem 1 and it will not be considered as a solution.**



2

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*General concepts*

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<i>b</i>								
<i>c</i>			X					
<i>d</i>	X	X			X		X	X
<i>e</i>								



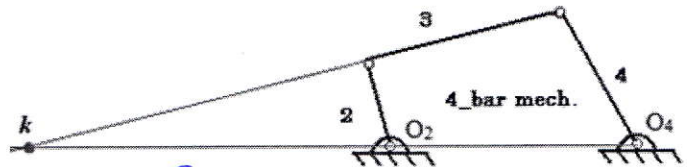
**Question 1.***continu*

**(8marks)**

1. A mechanism has 5 links will has a number of instantaneous centers equal

- a. 15                      b. 6                      c. 21                      **d. 10**                      e. none of the previous

2. For the four bar mechanism shown in the fig. if  $\omega_2 = 2400$  RPM,  $O_2k = 40$ cm and  $O_4k = 80$ cm. find the angular speed of link 4 ( $\omega_4$ ) in RPM



- a. 600                      b. 150                      c. 300                      **d. 1200**                      e. none of the previous

3. A rigid bar link has length equal 50cm and rotates about one of its joints by angular speed equal 400 rad/s. its tangential velocity in m/s will equal

- a. 100                      b. 50                      **c. 200**                      d. 25                      e. none of the previous

4. a 1.5m bar link rotate about one of its joints at a constant angular speed equal 50 rad/s. then the magnitude of its acceleration equal in  $m/s^2$ :

- a. 3750**                      b. 1350                      c. 150                      d. 600                      e. none of the previous

5. a bar link is analyzed statically. If the force at one of its joints is  $-7i + 5j$ , the Cartesian vector for the force in the other joint will be

- a.  $-5i + 7j$                       b.  $5i - 7j$                       c.  $-7i + 5j$                       **d.  $7i - 5j$**                       e. none of the previous

The flowing data are for questions 6 and 7:

When a cam rotates with speed equal 600 RPM from  $0^\circ$  to  $90^\circ$  it makes the follower to raise in SHM from 0cm to 4cm.

6. Find the follower displacement at cam angle equal  $75^\circ$ .

- a. 3.72 cm**                      b. 3.00 cm                      c. 2.00 cm                      d. 1.00 cm                      e. none of the previous

7. Find the follower velocity at cam angle equal  $75^\circ$ . round the solution to two decimal digits.

- a. 3.15 m/s                      b. 2.51 m/s                      c. 2.17 m/s                      **d. 1.26 m/s**                      e. none of the previous

8. An eccentric cam has eccentricity ( $e$ ) = 10 cm, find the displacement at angle =  $300^\circ$ .

- a. 15.0 cm                      b. 20.0 cm                      c. 13.4 cm                      **d. 5.0 cm**                      e. none of the previous

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Form 1



3

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<i>a</i>	X				X			X
<i>b</i>			X					
<i>c</i>		X					X	
<i>d</i>				X		X		
<i>e</i>								



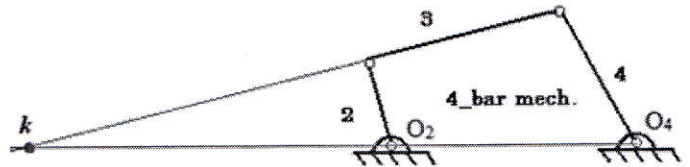
**Question 1:** *continu*

**(8marks)**

1. A mechanism has 6 links will has a number of instantaneous centers equal

- a. 15                      b. 6                      c. 21                      d. 10                      e. none of the previous

2. For the four bar mechanism shown in the fig. if  $\omega_2 = 600$  RPM,  $O_2k = 40$ cm and  $O_4k = 80$ cm. find the angular speed of link 4 ( $\omega_4$ ) in RPM



- a. 600                      b. 150                      c. 300                      d. 1200                      e. none of the previous

3. A rigid bar link has length equal 50cm and rotates about one of its joints by angular speed equal 100 rad/s. its tangential velocity in m/s will equal

- a. 100                      b. 50                      c. 200                      d. 25                      e. none of the previous

4. a 1.5m bar link rotate about one of its joints at a constant angular speed equal 20 rad/s. then the magnitude of its acceleration equal in  $m/s^2$ :

- a. 3750                      b. 1350                      c. 150                      d. 600                      e. none of the previous

5. a bar link is analyzed statically. If the force at one of its joints is :  $5i - 7j$ , the Cartesian vector for the force in the other joint will be

- a.  $-5i + 7j$                       b.  $5i - 7j$                       c.  $-7i + 5j$                       d.  $7i - 5j$                       e. none of the previous

The flowing data are for questions 6 and 7:

When a cam rotates with speed equal 600 RPM from  $0^\circ$  to  $90^\circ$  it makes the follower to raise in SHM from 0cm to 4cm.

6. Find the follower displacement at cam angle equal  $30^\circ$ .

- a. 3.72 cm                      b. 3.00 cm                      c. 2.00 cm                      d. 1.00 cm                      e. none of the previous

7. Find the follower velocity at cam angle equal  $30^\circ$  .round the solution to two decimal digits.

- a. 3.15 m/s                      b. 2.51 m/s                      c. 2.17 m/s                      d. 1.26 m/s                      e. none of the previous

8. An eccentric cam has eccentricity (e) = 10 cm, find the displacement at angle =  $120^\circ$ .

- a. 15.0 cm                      b. 20.0 cm                      c. 13.4 cm                      d. 5.0 cm                      e. none of the previous

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**Question 1:**

**(8marks)**

*General concepts*

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<i>a</i>			X					
<i>b</i>		X		X		X		
<i>c</i>	X				X		X	X
<i>d</i>								
<i>e</i>								



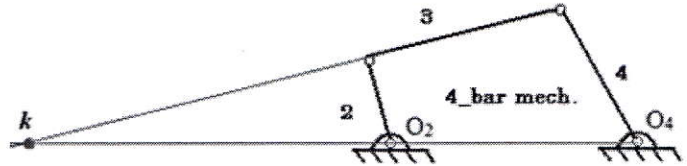
**Question 1:** *continu*

**(8marks)**

1. A mechanism has 7 links will has a number of instantaneous centers equal

- a. 15                      b. 6                      **c. 21**                      d. 10                      e. none of the previous

2. For the four bar mechanism shown in the fig. if  $\omega_2 = 300$  RPM,  $O_2k = 40$ cm and  $O_4k = 80$ cm. find the angular speed of link 4 ( $\omega_4$ ) in RPM



- a. 600                      **b. 150**                      c. 300                      d. 1200                      e. none of the previous

3. A rigid bar link has length equal 50cm and rotates about one of its joints by angular speed equal 200 rad/s. its tangential velocity in m/s will equal

- a. 100**                      b. 50                      c. 200                      d. 25                      e. none of the previous

4. a 1.5m bar link rotate about one of its joints at a constant angular speed equal 30 rad/s. then the magnitude of its acceleration equal in  $m/s^2$ :

- a. 3750                      **b. 1350**                      c. 150                      d. 600                      e. none of the previous

5. a bar link is analyzed statically. If the force at one of its joints is  $:7i - 5j$ , the Cartesian vector for the force in the other joint will be

- a.  $-5i + 7j$                       b.  $5i - 7j$                       **c.  $-7i + 5j$**                       d.  $7i - 5j$                       e. none of the previous

The flowing data are for questions 6 and 7:

When a cam rotates with speed equal 600 RPM from  $0^\circ$  to  $90^\circ$  it makes the follower to raise in SHM from 0cm to 4cm.

6. Find the follower displacement at cam angle equal  $60^\circ$ .

- a. 3.72 cm                      **b. 3.00 cm**                      c. 2.00 cm                      d. 1.00 cm                      e. none of the previous

7. Find the follower velocity at cam angle equal  $60^\circ$ . round the solution to two decimal digits.

- a. 3.15 m/s                      b. 2.51 m/s                      **c. 2.17 m/s**                      d. 1.26 m/s                      e. none of the previous

8. An eccentric cam has eccentricity ( $e$ ) = 10 cm, find the displacement at angle =  $250^\circ$ .

- a. 15.0 cm                      b. 20.0 cm                      **c. 13.4 cm**                      d. 5.0 cm                      e. none of the previous

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**Question 2:**

**(6 marks)**

Consider the following loop closure equation for four bar mechanism and the given data in the table:

$$d_1 U_0 + d_4 U_{\theta_4} = d_3 U_{\theta_3} + d_2 U_{\theta_2}$$

Qunt.	$d_1$	$d_2$	$d_3$	$d_4$	$\theta_2$	$\theta_3$	$\theta_4$	$\omega_2$	$\omega_3$	$\omega_4$
value	0.9	0.2	0.7	0.5	60	25	110	300	?	?
Unit	m	m	m	m	Degree	Degree	Degree	RPM	RPM	RPM

1. Derive an expression to find  $\omega_3$  and  $\omega_4$
2. Substitute the values in table to find  $\omega_3$  and  $\omega_4$

$$1. \quad d_2 \omega_2 \dot{U}_{\theta_2} + d_3 \omega_3 \dot{U}_{\theta_3} = d_4 \omega_4 \dot{U}_{\theta_4}$$

Dot product both sides by  $U_{\theta_3}$  to eliminate  $\omega_3$

$$d_2 \omega_2 \sin(\theta_3 - \theta_2) + 0 = d_4 \omega_4 \sin(\theta_3 - \theta_4)$$

Solve for  $\omega_4$ : 
$$\omega_4 = \frac{d_2 \omega_2 \sin(\theta_3 - \theta_2)}{d_4 \sin(\theta_3 - \theta_4)}$$

To find  $\omega_3$ , dot product both sides of derivative equation by  $U_{\theta_4}$  to eliminate  $\omega_4$ :

$$d_2 \omega_2 \sin(\theta_4 - \theta_2) + d_3 \omega_3 \sin(\theta_4 - \theta_3) = 0$$

Solve this equation for  $\omega_3$ : 
$$\omega_3 = -\frac{d_2 \omega_2 \sin(\theta_4 - \theta_2)}{d_3 \sin(\theta_4 - \theta_3)}$$

2. substitute the values from table

$$\omega_3 = -\frac{(0.2)(300)\sin(110 - 60)}{0.7 \sin(110 - 25)} = -65 \text{ RPM} \quad \approx -6.9 \frac{\text{Rad}}{\text{s}}$$

$$\omega_4 = \frac{(0.2)(300)\sin(25 - 60)}{0.5 \sin(25 - 110)} = 69 \text{ RPM} \quad \approx 7.22 \frac{\text{Rad}}{\text{s}}$$

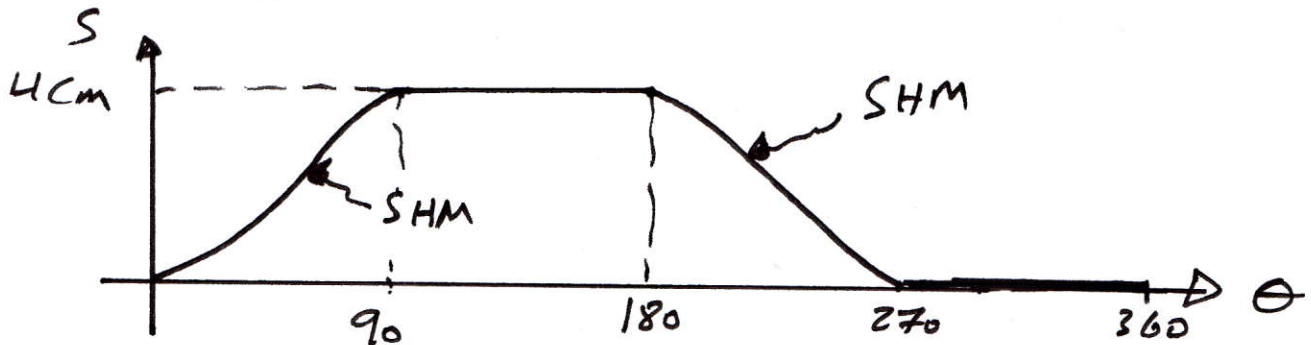
**Question 3:**

**(6 marks)**

Assume a cam has basic radius of 6cm and has the following follower program

- $0 \rightarrow 90^\circ$  : SHM rise to 4cm
- $90^\circ \rightarrow 180^\circ$  : Dwell
- $180^\circ \rightarrow 270^\circ$  : SHM return to 0
- $270^\circ \rightarrow 360^\circ$  : Dwell

1. Draw an approximate sketch for S- $\theta$  diagram



2. Complete the following table and show all the calculations you used to find these values

For SHM:  $s(\theta) = \frac{H}{2} \left( 1 - \cos\left(\frac{\pi\theta}{\beta}\right) \right)$

For 45 degree:  $H = 4\text{cm}$ ,  $\beta = 90^\circ$ ,  $\Theta = 45$

$s(45) = \frac{0.04}{2} \left( 1 - \cos\left(\frac{\pi 45}{90}\right) \right) = 0.02\text{m}$

$r_c = r_b + s = 2\text{cm} + 10\text{cm} = 12\text{cm}$

For 225 degree:  $H = 4\text{cm}$ ,  $\beta = 90^\circ$ ,  $\Theta = 225 - 180 = 45$

$s(225) = 4 - \frac{0.04}{2} \left( 1 - \cos\left(\frac{\pi 45}{90}\right) \right) = 0.02\text{m}$

$r_c = r_b + s = 2\text{cm} + 10\text{cm} = 12\text{cm}$

Cam angle (degree)	Follower disp. (m)	Cam radius (cm)
45	0.02	12.8
225	0.02	12.8

### Theory of machines data sheet

Student name: \_\_\_\_\_ Reg. No: \_\_\_\_\_

$$\begin{aligned} s &= 2H\left(\frac{\theta}{\beta}\right)^2; \quad 0 < \theta < \beta/2 & s &= H\left(1 - 2\left(1 - \frac{\theta}{\beta}\right)^2\right); \quad \beta/2 < \theta < \beta \\ \dot{s} &= 4H\omega\left(\frac{\theta}{\beta}\right); \quad 0 < \theta < \beta/2 & \dot{s} &= 4H\frac{\omega}{\beta}\left(1 - \frac{\theta}{\beta}\right); \quad \beta/2 < \theta < \beta \\ \ddot{s} &= 4H\left(\frac{\omega}{\beta}\right)^2; \quad 0 < \theta < \beta/2 & \ddot{s} &= -4H\left(\frac{\omega}{\beta}\right)^2; \quad \beta/2 < \theta < \beta \end{aligned}$$

$$s(\theta) = \frac{H}{2} \left( 1 - \cos\left(\frac{\pi\theta}{\beta}\right) \right)$$

$$\dot{s}(\theta) = \frac{H}{2} \left( \frac{\pi\omega}{\beta} \right) \sin\left(\frac{\pi\theta}{\beta}\right)$$

$$\ddot{s}(\theta) = \frac{H}{2} \left( \frac{\pi\omega}{\beta} \right)^2 \cos\left(\frac{\pi\theta}{\beta}\right)$$

$$s(\theta) = \frac{h}{\pi} \left( \frac{\pi\theta}{\beta} - \frac{1}{2} \sin\left(\frac{2\pi\theta}{\beta}\right) \right)$$

$$\dot{s}(\theta) = \frac{h}{\pi} \left( \frac{\omega}{\beta} \right) \left( 1 - \cos\left(\frac{2\pi\theta}{\beta}\right) \right)$$

$$\ddot{s}(\theta) = 2h\pi \left( \frac{\omega}{\beta} \right)^2 \sin\left(\frac{2\pi\theta}{\beta}\right)$$

$$h = e\{1 - \cos(\theta)\} \quad v = \omega e \sin(\theta) \quad a = \omega^2 e \cos(\theta)$$